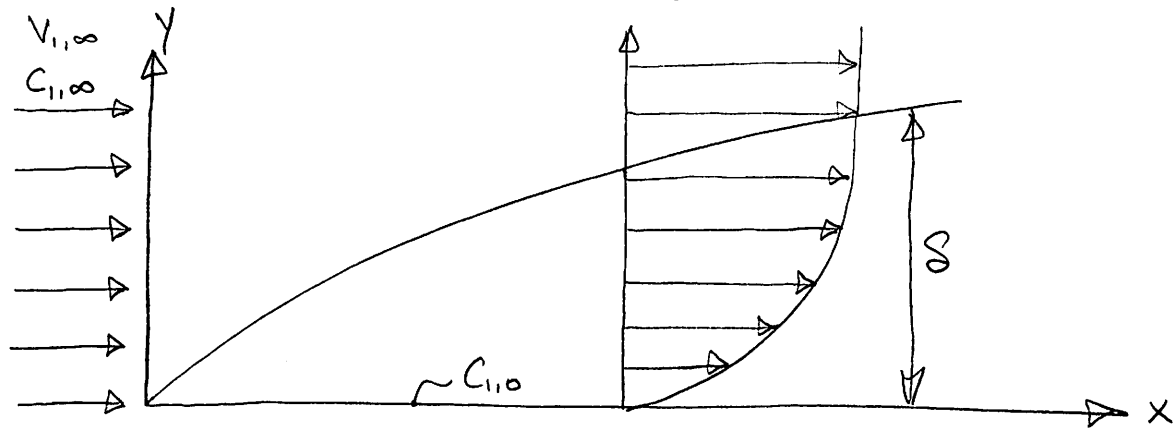
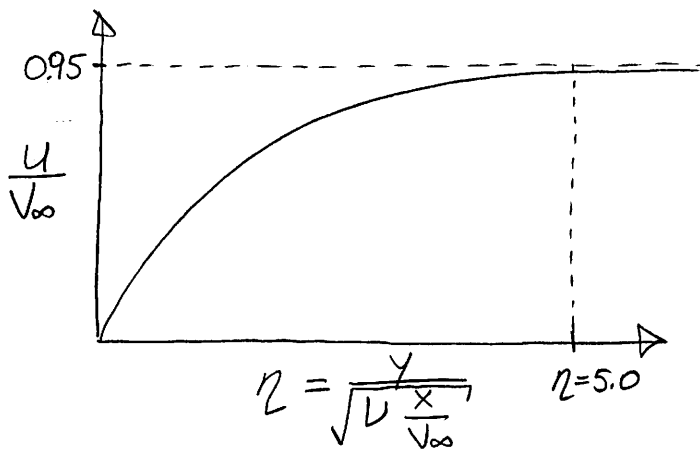


Convective Mass Transfer (External)

If we examine air blowing over a flat plate



We've already solved this problem:

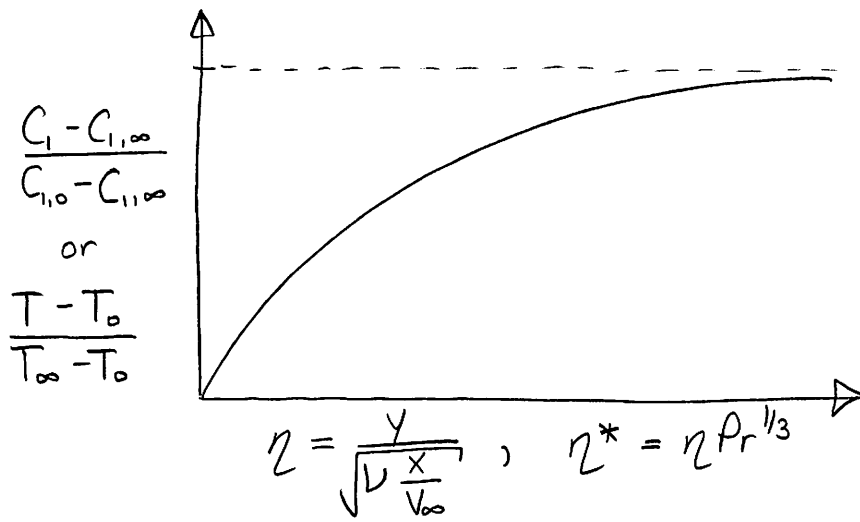


⇒ See page 84 of notes.

Now if we look at the mass transfer analogy (N-S eqn.)

$$u \frac{\partial C_1}{\partial x} + v \frac{\partial C_1}{\partial y} = D_{1,g} \frac{\partial^2 C_1}{\partial y^2}$$

Note, this is the exact same equation as in heat transfer, so we can make a really simple analogy and our mass transfer results will be the same



⇒ See page 89 of notes

For the heat transfer case: $Pr = \frac{\nu}{\alpha}$

For the mass transfer case: $Sc = \frac{\nu}{D_{1,2}}$

Schmidt number
 ν = kinematic viscosity
 $D_{1,2}$ = diffusivity (mass)

Our solutions become:

$$Nu = \frac{hx}{k} = 0.332 Re_x^{1/2} Pr^{1/3}$$

$$\overline{Nu}_L = \frac{\bar{h}L}{k} = 0.664 Re_L^{1/2} Pr^{1/3}$$

For mass transfer:

$$Sh = \frac{h_m x}{D} = 0.332 Re_x^{1/2} Sc^{1/3} \quad \text{Sherwood number}$$

We can now develop a Colburn analogy for mass transfer to express the mass transfer coefficient in terms of friction factor, f : (pg. 90 of notes)

$$C_{f,x} = \frac{\tau}{\frac{1}{2} \rho V_\infty^2} = \frac{2a_2}{\sqrt{Re_x}}, \quad a_2 = 0.332$$

$$Nu_x = a_2 Re_x^{1/2} Pr^{1/3} \quad (\text{Flat plate})$$

$$\left(\frac{Nu_x}{Re_x Pr} \right) = \frac{1}{2} \frac{2a_2}{\sqrt{Re_x}} \frac{1}{Pr^{2/3}} = \frac{1}{2} C_{f,x} \frac{1}{Pr^{2/3}}$$

$$St = \frac{h}{\rho C_p V_\infty} \quad (\text{Stanton number}) \equiv \frac{\text{heat transferred to a fluid}}{\text{thermal capacity of the fluid}}$$

So our heat transfer analogy was:

$$St \cdot Pr^{2/3} = \frac{C_{f,x}}{2} \Rightarrow \text{See page 90 of notes for more.}$$

For mass transfer, we have:

$$\left(\frac{Sh_x}{Re_x Sc} \right) = \frac{1}{2} \frac{2a_2}{\sqrt{Re_x}} \frac{1}{Sc^{2/3}} = \frac{1}{2} C_{f,x} \frac{1}{Sc^{2/3}} = St_m$$

By analogy (you can prove it)

$$\boxed{St_m \cdot Sc^{2/3} = \frac{C_{f,x}}{2}} \quad \text{Mass transfer Colburn analogy}$$

$$\boxed{St_m = \frac{h_m}{V_\infty}} \quad \text{Mass transfer Stanton number}$$

Notice that we can combine the two analogies

$$St \cdot Pr^{2/3} = St_m \cdot Sc^{2/3}$$

$$\frac{h}{\rho C_p V_\infty} Pr^{2/3} = \frac{h_m}{V_\infty} Sc^{2/3}$$

$$\boxed{\frac{h}{h_m} = (\rho C_p) \left(\frac{\alpha}{D_{1,2}} \right)^{2/3} = \rho \cdot C_p \cdot Le^{2/3}}, \quad \text{where } \boxed{Le = \frac{\alpha}{D_{1,2}}}$$

\equiv Lewis number

Note this means we can measure mass transfer (h_m) and use it to back calculate heat transfer (h) \Rightarrow very powerful

Example | Dry air at atmospheric pressure blows across a thermometer which is enclosed in a damp cover. This is a classical wet bulb measurement. The temperature reads 18.3°C . What is the dry air temperature, T_∞ ?

The heat needed to evaporate the water comes from the surrounding air (at steady state).

$$hA(T_\infty - T_w) = \dot{m}_w h_{fg} \quad (1) \Rightarrow \text{Heat transfer}$$

$$\dot{m}_w = h_m A (p_w - p_\infty) \quad (2) \Rightarrow \text{Mass transfer}$$

Combining (1) and (2) Mass concentrations

$$hA(T_\infty - T_w) = h_m A (p_w - p_\infty) h_{fg}$$

Using our Colburn analogies

$$p_a C_p Le^{2/3} (T_\infty - T_w) = (p_w - p_\infty) h_{fg} \quad (3)$$

We know C_w corresponds to the saturation conditions at T_w
From steam tables:

$$p_{\text{sat}} = 2107 \text{ Pa at } T_{\text{sat}} = 18.3^\circ\text{C}$$

$$p_w = \frac{p_{\text{sat}}}{R_w T_w} = \frac{(2107)(18)}{(8315)(291.3)} = 0.01566 \text{ kg/m}^3$$

For the other properties:

$$p_\infty = 0 \text{ (dry air)}$$

$$p_a = \text{air density} = \frac{\rho}{RT} = \frac{101325}{(287)(291.3)} = 1.212 \text{ kg/m}^3$$

$$C_p = 1.004 \text{ kJ/kg}$$

$$\frac{\alpha}{D_{1,2}} = Le = \frac{Sc}{Pr} = 0.845$$

$$h_{fg} = 2.456 \text{ MJ/kg}$$

Back substituting into equation ③

$$T_\infty - T_w = \frac{(0.01566 - 0)(2.456 \times 10^6)}{(1.212)(1004)(0.845)^{2/3}} = 35.36^\circ\text{C}$$

$$\Rightarrow \boxed{T_\infty = 53.69^\circ\text{C}} \Rightarrow \text{Dry bulb temperature}$$

Mass Transfer Summary

Heat Transfer	Mass Transfer
h [W/m ² ·K]	h_m [m/s]
Re [dimensionless]	Re
$Nu = \frac{hL}{k_f} \equiv \text{Nusselt}$	$Sh = \frac{h_m L}{D_{1,2}} \equiv \text{Sherwood}$
$Nu = f(Re, Pr)$	$Sh = f(Re, Sc)$
$Pr = \frac{\nu}{\alpha} \equiv \text{Prandtl}$	$Sc = \frac{\nu}{D_{1,2}} \equiv \text{Schmidt}$
$Ra = \frac{g\beta\Delta TL^3}{\nu\alpha} \equiv \text{Rayleigh}$	$Ra_m = \frac{g\Delta\rho L^3}{\mu D_{1,2}} \equiv \text{Rayleigh mass transf. (Natural conv.)}$
	$Le = \frac{\alpha}{D_{1,2}}$